

# DOS : MDOF systems

Number of participants: 15




1. **The mode shapes and eigenfrequencies of a system are determined by (K and M are the stiffness and mass matrices)**

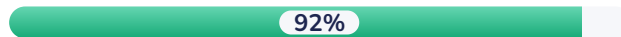
**11 correct answers**  
out of 12 respondents

Calculating the eigenvalues of the stiffness matrix K of the system



1 vote


 solving a generalized eigenvalue problem of the type  $(K - \omega^2 M) \{\Phi_i\} = 0$



11 votes

Calculating the eigenvalues of the mass matrix M of the system



0 votes

Calculating  $\omega = \sqrt{K/M}$



0 votes



## 2. If a system has $n$ degrees of freedom, it has

**12 correct answers**  
out of 12 respondents

$2n$  natural frequencies

0%

0 votes



$n$  natural frequencies

100%

12 votes

$(n + \text{the number of excitations})$  natural frequencies

0%

0 votes

it depends on the type of system

0%

0 votes



## 3. The mode shapes are orthogonal with respect to the

**6 correct answers**  
out of 10 respondents



stiffness matrix

100%

10 votes



mass matrix

60%

6 votes

damping matrix

0%

0 votes



#### 4. The interest of projecting the equations of motion in the modal domain is to:

2 correct answers  
out of 13 respondents



potentially reduce the number of equations to solve



5 votes



decouple the equations of motion and facilitate solving them



10 votes

work with physical quantities for a better understanding of the system's behavior



6 votes



#### 5. Which of these quantities is a global quantity for a given structure (i.e does not change with the position where you measure the response)

5 correct answers  
out of 13 respondents



the eigenfrequency



9 votes

the anti-resonance frequency



3 votes



the damping coefficient



9 votes



## 6. An anti-resonance happens when

**12 correct answers**  
out of 12 respondents

The contribution to the response of the two close modes as equal amplitude and equal phase

0%

0 votes

The contribution to the response of the two close modes has equal amplitude and opposite phase



100%

12 votes

The displacement is zero for the two closest modes

0%

0 votes



## 7. What kind of hypothesis can be made on the damping matrix to decouple the equations of motion in the modal domain ?

**1 correct answer**  
out of 10 respondents

Rayleigh Damping

100%

10 votes

Lagrange Damping

0%

0 votes

Modal damping

50%

5 votes

Viscous damping

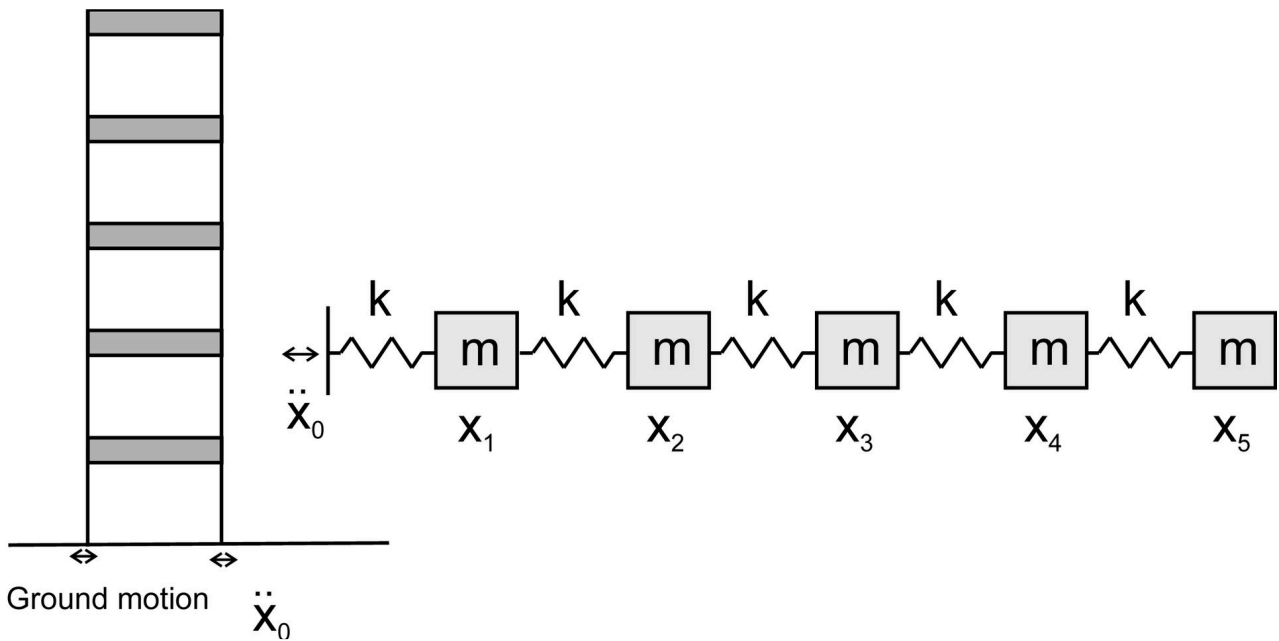
30%

3 votes



**8. How many mode shapes and eigenfrequencies does this building simplified model have ?**

**10 correct answers**  
out of 14 respondents



one

0%

0 votes

three

14%

2 votes



five

71%

10 votes

an infinity

14%

2 votes



**9. If this system is excited with an harmonic force applied to the bottom mass, whose frequency is close to the first natural frequency, the motion will correspond to**

**10 correct answers**  
out of 13 respondents

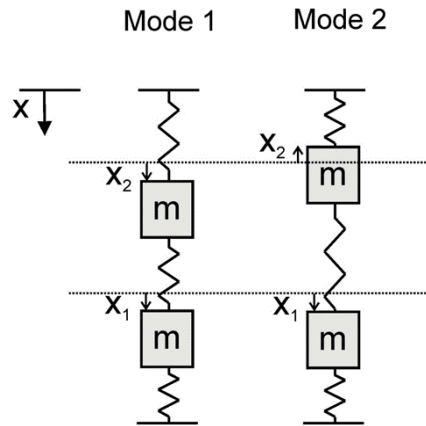
**2 DOFs system mode shapes**

$$\omega_1^2 = k/m$$

$$\omega_2^2 = 3k/m$$

$$\psi_1 = \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$$

$$\psi_2 = \begin{Bmatrix} 1 \\ -1 \end{Bmatrix}$$



14



The first mode shape where the two masses move in phase



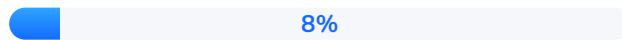
10 votes

The second mode shape where the two masses move out of phase



2 votes

A combination of the two modeshapes



1 vote



10.

**If this system is excited with an harmonic force applied to the bottom mass, whose frequency is the average of the first and second natural frequencies of the system, the motion will correspond to**

**8 correct answers**  
out of 10 respondents

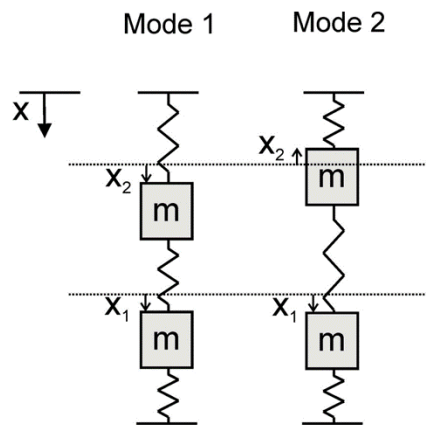
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14

The first mode shape where the two masses move in phase



0 votes

The second mode shape where the two masses move out of phase



2 votes



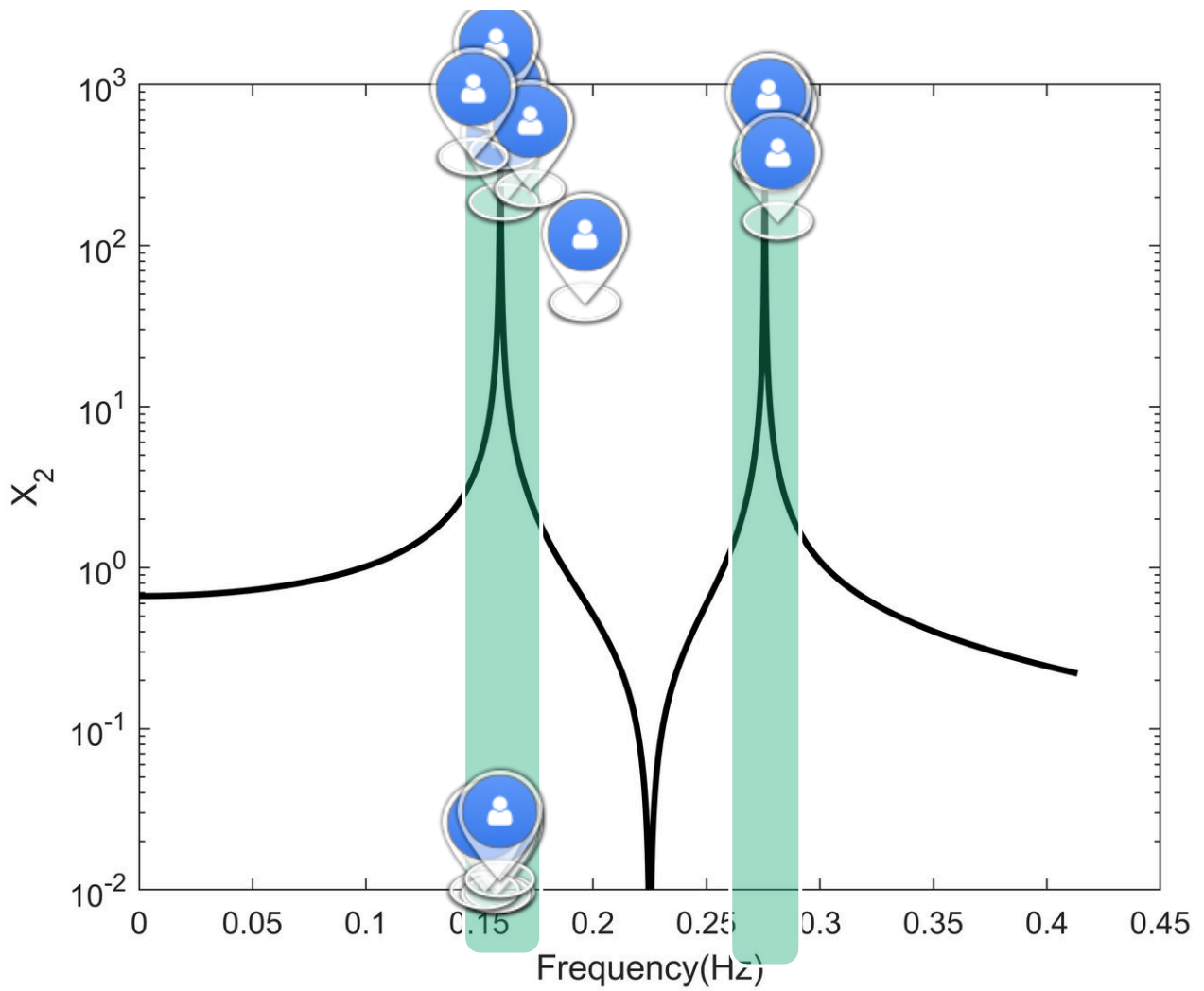
A combination of the two mode shapes



8 votes

**11. Where are the resonant frequencies of the 2 DOFs system on this graph ?**

13 respondents

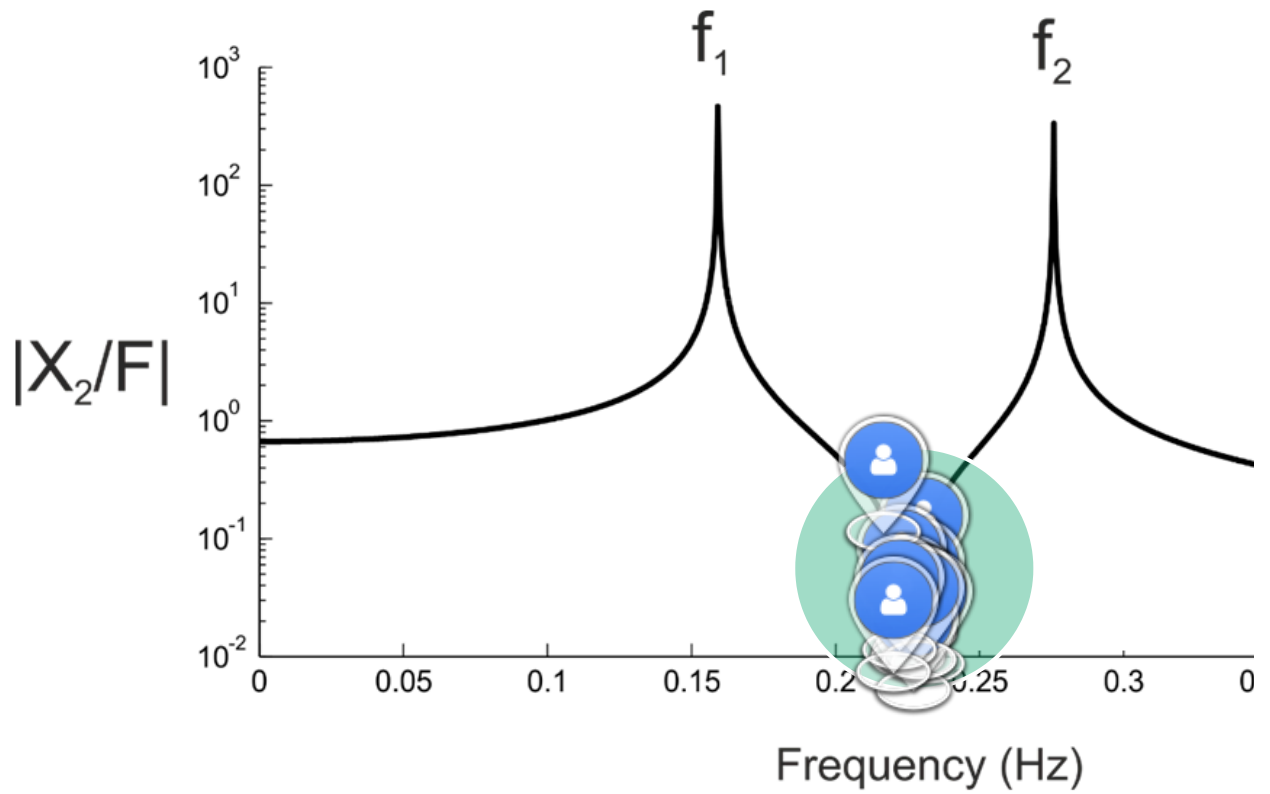






**12. Where is the anti-resonance of the 2 DOFs system on this graph ?**

12 respondents

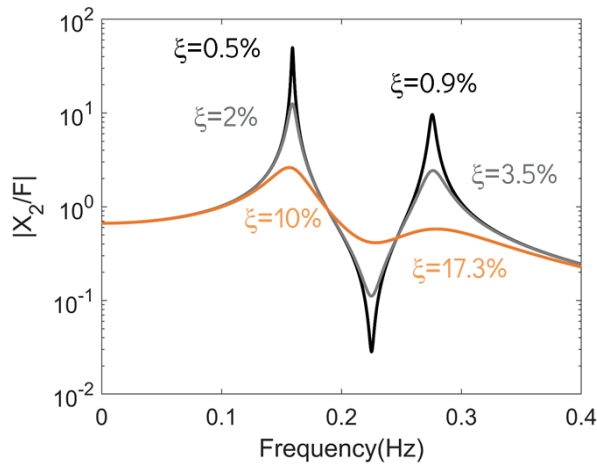




**13. Why is the damping coefficient higher for the second mode than for the first mode for the damped two dofs system treated in the examples of the course ?**

9 respondents

Example of a 2 DOFs system



$k=1 \text{ N/m}$ ,  $m=1\text{kg}$ ,

$b=0.01 \text{ N/ms}$

$b=0.04 \text{ N/ms}$

$b=0.2 \text{ N/ms}$

37

Because in the first mode the 2 mass are in phase instead in the seconde mode they are in opposition of phase

.

The damping if fonction of the frequency The movements is bigger between the two mass

Second Natural frequency higher

Eigenvalues of damping coefficient are closer to the eigenvalues of the second mode shape

Because the spring between the 2 masses is not extended

The masses go in opposite directions

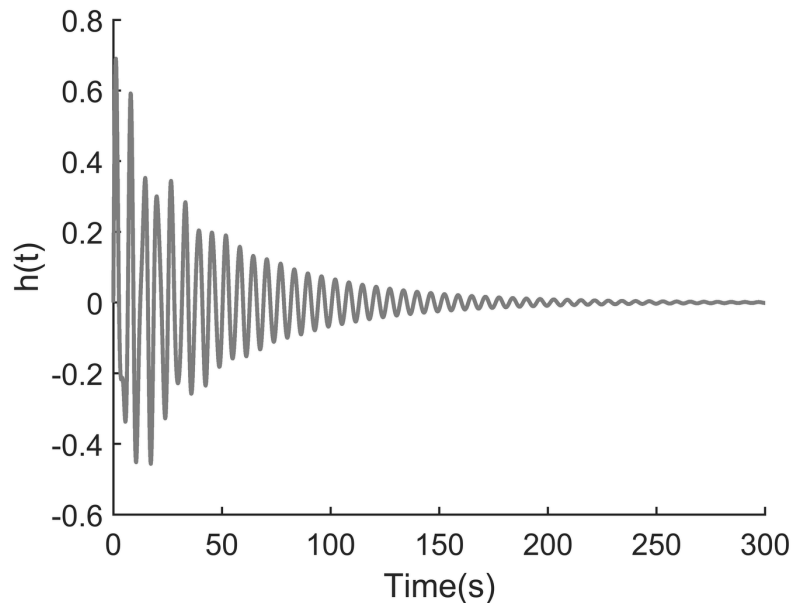
It's giroup

Because in the second mode, the middle spring dissipates energy



## 14. How would you extract the first natural frequency from the impulse response ?

0 correct answer  
out of 5 respondents



Two first same pics at the bottom (0.4)

Fourier analysis or using the exponential function decreasing

See the frequency at steady state

Determining the exponential envelope after the zone of interaction

Put a low pass filter Maybe the first frequency because it is not attenuated

### Correct answer

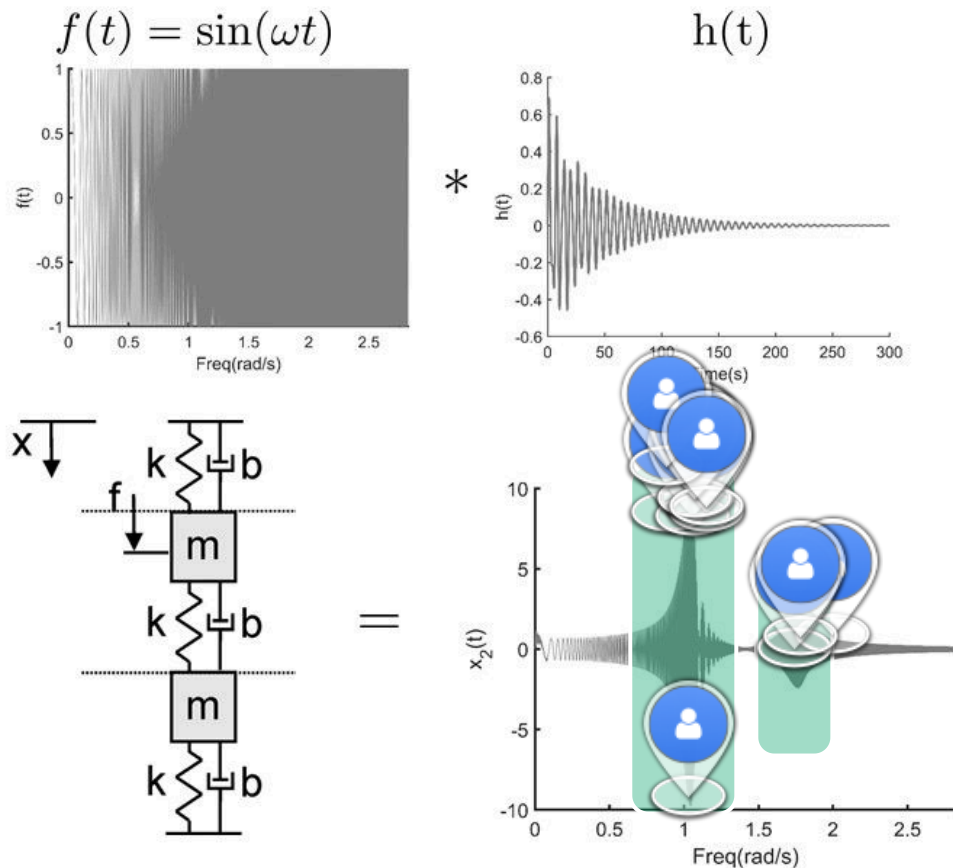
**By looking at the end of the response and extracting the period from the distance between two crossings of the axis.**



**15. This is the time domain response of a damped 2 DOFs system under sine sweep excitation. Where do you see the resonances on the time domain response ?**

11 respondents

## Sine sweep excitation





## 16. When a system is excited by its base, it is easier to write the unknown displacements

4 correct answers  
out of 8 respondents

as relative displacement between the neighbouring DOFS



4 votes



as relative displacement between the base and each DOF.



4 votes

in mm instead of m



0 votes



## 17. When doing so, the equation of motion is equivalent to the case of an applied force which is

5 correct answers  
out of 8 respondents



proportional to the applied acceleration



5 votes

inversely proportional to the applied acceleration



2 votes

proportional to the applied displacement



1 vote