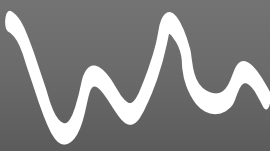
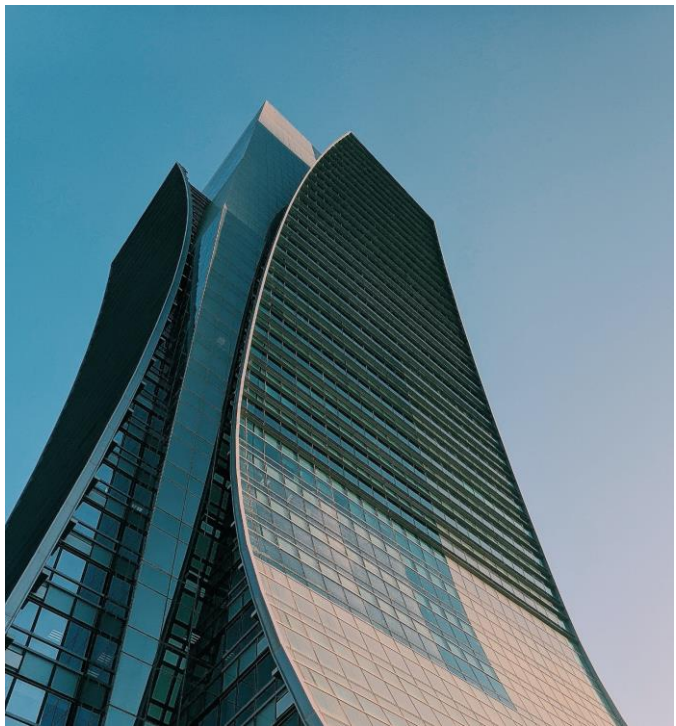
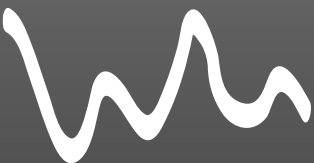


FINITE ELEMENT MODELS



1

COMPLEX STRUCTURES



2

Complex structures in mechanical engineering



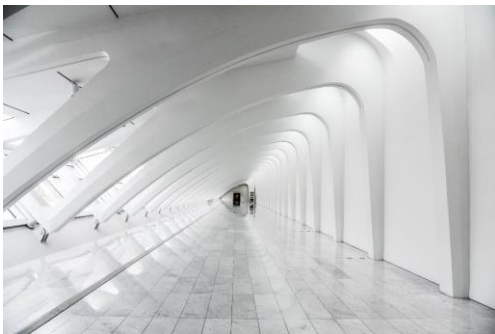
3D kinematics



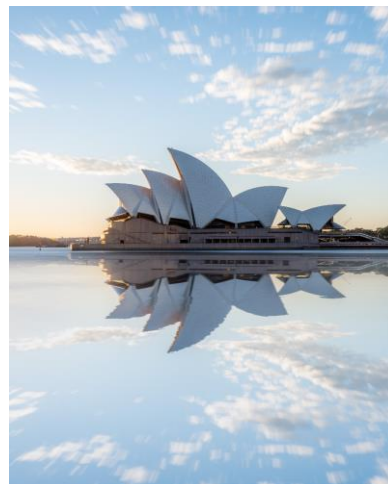
3

3

Complex structures in civil engineering



3D kinematics



4

4

Finite element models of complex structures

30 storey building model

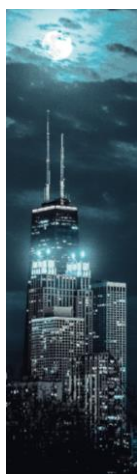
Beam and plate elements

- 9222 nodes
- 11700 elements
- 55332 degrees of freedom



$$M\ddot{x} + Kx = f$$

55332 equations with 55332 unknowns



SDA Tools
VIBRATION SOFTWARE & CONSULTING



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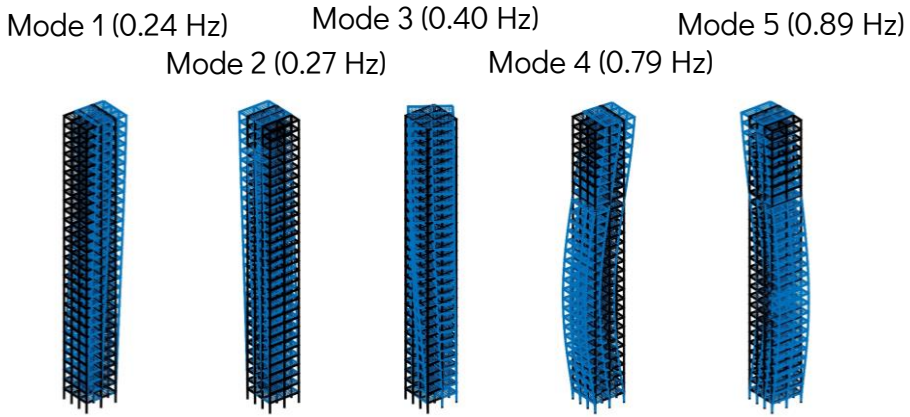
5

MODE SHAPES



6

Building mode shapes



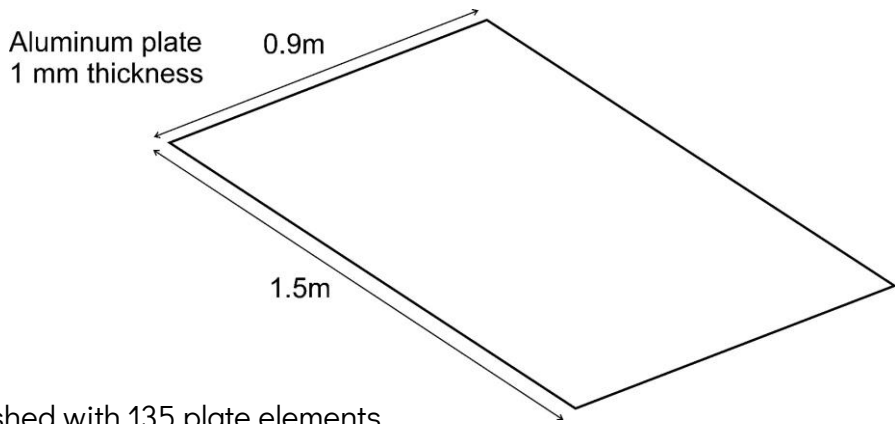
Mode shapes are generally 'mass normalized' : $\mu_i = 1$

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Cantilever plate

Cantilever plate

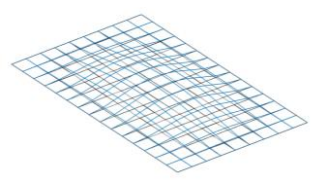


8

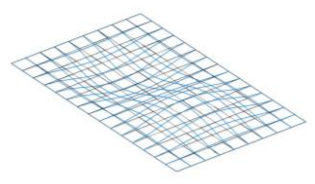
8

Cantilever plate mode shapes

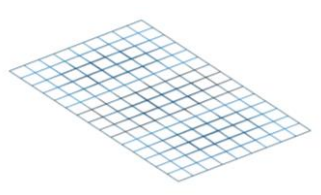
Mode 1 (8.2 Hz)



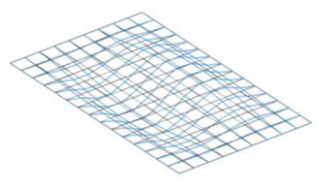
Mode 2 (11.81 Hz)



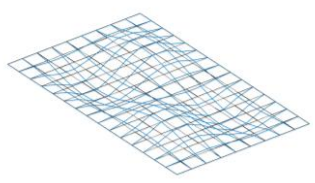
Mode 3 (18.49 Hz)



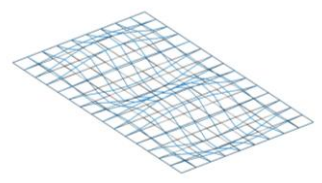
Mode 4 (22.11 Hz)



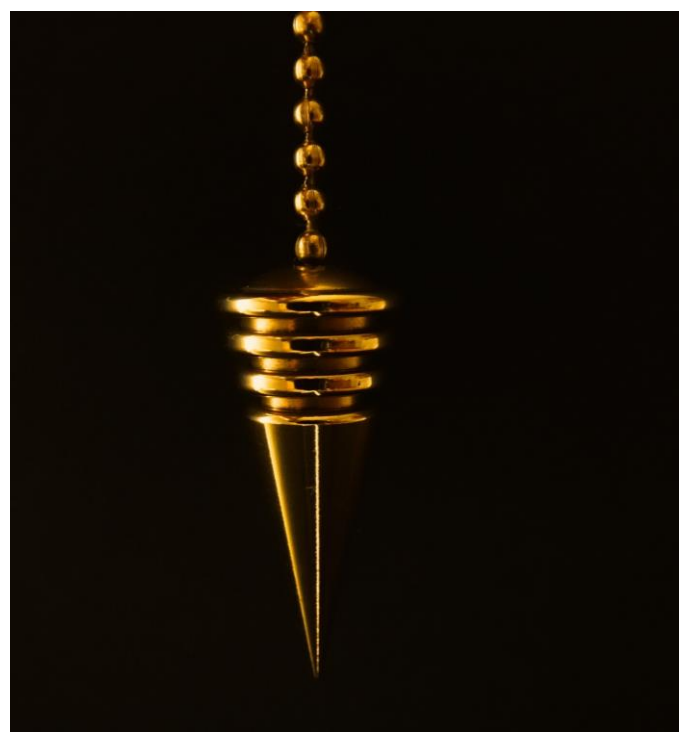
Mode 5 (25.38 Hz)



Mode 6 (28.68 Hz)



HARMONIC RESPONSE



Computation of the dynamic response

$$M\ddot{x} + Kx = f$$

Orthogonality conditions Projection in the modal basis

$$\begin{aligned} \psi_i^T M \psi_j &= \delta_{ij} \mu_i \\ \psi_i^T K \psi_j &= \delta_{ij} \mu_i \omega_i^2 \end{aligned} \quad \longrightarrow \quad \boxed{\mu_i \ddot{z}_i + \mu_i \omega_i^2 z_i = F_i}$$

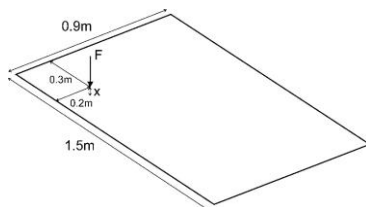
- In practice, the number of dofs in the finite element model is dictated by the details of the **geometry** and for large models, it is not possible to compute all the modeshapes.
- In addition, the **number of modes** in the frequency band of interest is usually quite low (i.e 10 to 50 modes)

→ Important reduction when projecting on the modal basis

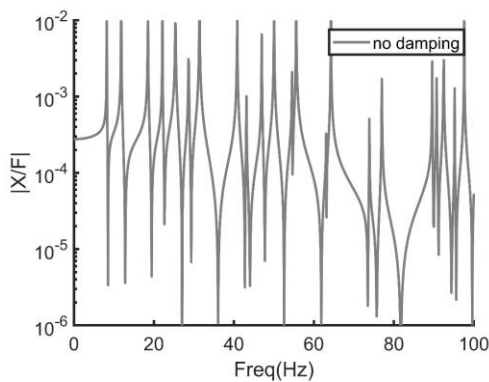
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Harmonic response of the cantilever plate



$$(K - \omega^2 M)X = F$$



$$X(\omega) = \sum_{i=1}^n \frac{\psi_i^T F \psi_i}{\mu_i (\omega_i^2 - \omega^2)} \quad n=25$$

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Global damping models

$$M\ddot{x} + C\dot{x} + Kx = f$$

Global damping models

Rayleigh damping

$$C = \alpha K + \beta M$$

Global viscous model

$$C = \alpha K$$

Loss factor – Hysteretic damping

$$(K + i\omega C - \omega^2 M)X = F$$

$$(K(1 + i\eta) - \omega^2 M)X = F \quad C = \frac{\eta}{\omega} K$$

→ Used most of the time for frequency domain computations

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Global damping models

For proportional (global) damping models

$$\mu_i \ddot{z}_i + 2\xi_i \mu_i \omega_i \dot{z}_i + \mu_i \omega_i^2 z_i = F_i$$

$$\text{Rayleigh damping} \Rightarrow \xi_i = \frac{1}{2} \left(\alpha \omega_i + \frac{\beta}{\omega_i} \right)$$

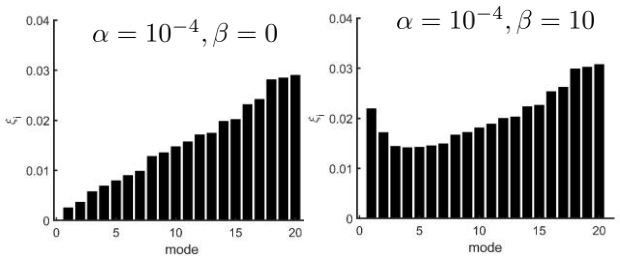
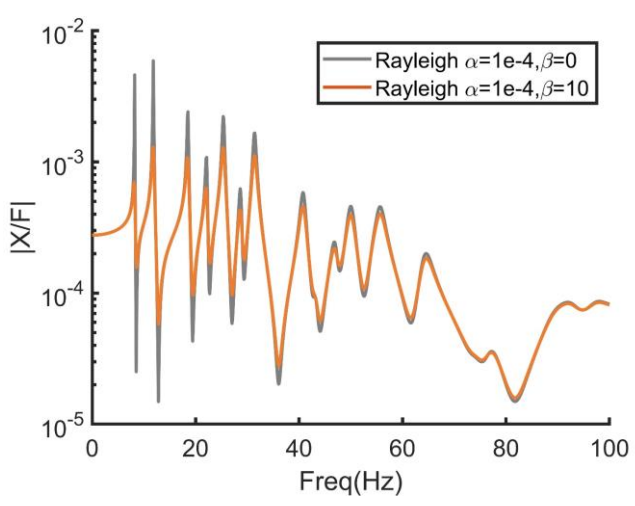
$$\text{Loss factor} \Rightarrow \xi_i = \frac{\eta}{2} \quad \text{Constant modal damping}$$

$$\text{Modal damping} \Rightarrow \xi_i \quad \text{For each mode}$$

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Harmonic response : comparison of global damping models

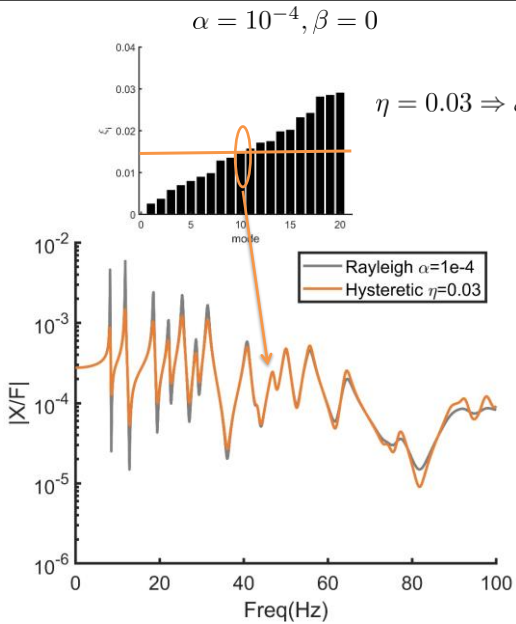


$$X(\omega) = \sum_{j=1}^n \frac{\psi_j^T F \psi_j}{\mu_j (\omega_j^2 - \omega^2 + 2i\xi_j \omega \omega_j)}$$

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Harmonic response : comparison of global damping models



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Local damping models

$$M\ddot{x} + C\dot{x} + Kx = f$$

Viscous damping

$$C_i = \alpha_i K_i \quad \text{In each substructure}$$

Loss factor – Hysteretic damping

$$E(1 + i\eta) \quad \text{Loss factor can be different for each material}$$

→ Non proportional damping

$$\Psi^T C \Psi \quad \text{is not diagonal}$$

If damping is small

$$\Rightarrow \xi_i = f(\alpha_i, \eta_i, \dots)$$

Usually identified experimentally, or from similar structures

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TIME DOMAIN
RESPONSE



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Base excitation : time domain response

$$M\ddot{x}_r + C\dot{x}_r + Kx_r = -M\ddot{x}_b \quad \ddot{x}_b = T\ddot{x}_0$$

$$\longrightarrow M\ddot{x}_r + C\dot{x}_r + Kx_r = -MT\ddot{x}_0$$

Projection in the modal basis using mode shapes with fixed base ($x_b = 0$)

$$x_r = \sum_{i=1}^n z_i(t)\psi_i \quad x_r = \Psi Z$$

$$\Psi^T M \Psi \ddot{z}_r + \Psi^T C \Psi \dot{z}_r + \Psi^T K \Psi z_r = -\Psi^T M T \ddot{x}_0$$

$$\mu_i \ddot{z}_{ri} + 2\mu_i \xi_i \omega_i \dot{z}_{ri} + \mu_i \omega_i^2 z_{ri} = -\Gamma_i \ddot{x}_0$$

$$\Gamma_i = \psi_i^T M T \quad \text{Modal acceleration factor}$$

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Base excitation : time domain response

$$\mu_i \ddot{z}_{ri} + 2\mu_i \xi_i \omega_i \dot{z}_{ri} + \mu_i \omega_i^2 z_{ri} = -\Gamma_i \ddot{x}_0 \quad \Gamma_i = \psi_i^T M T$$



Impulse response

$$h_i(t) = \frac{e^{-\xi_i \omega_i t}}{\mu_i \omega_{di}} \sin(\omega_{di} t) \quad \omega_{di} = \omega_i \sqrt{1 - \xi_i^2}$$

Convolution

$$z_i(t) = h_i(t) * -\Gamma_i \ddot{x}_0$$

Expansion

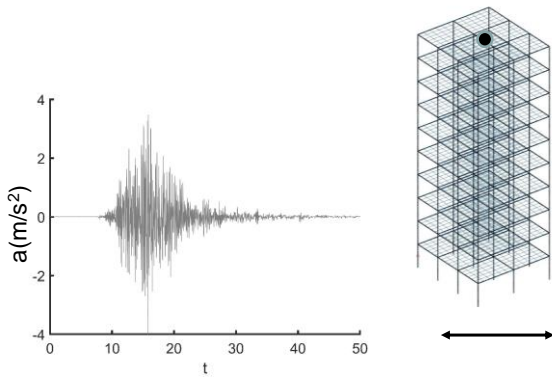
$$x_r(t) = \sum_{i=1}^n z_i(t)\psi_i$$

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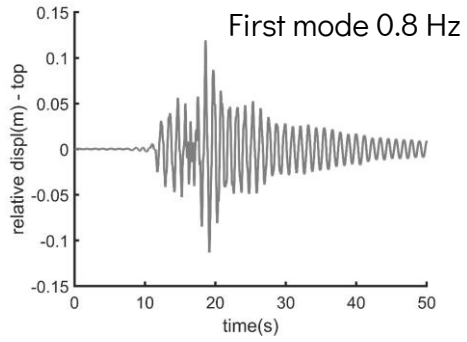
20

Earthquake response of a 10 storey building

Modal damping / 20 modes (0-15 Hz)
 $\xi = 0.01$



Ground motion (acceleration)

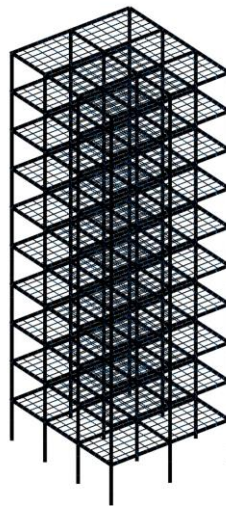
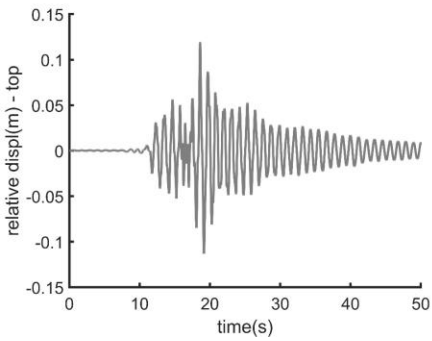


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Earthquake response of a 10 storey building

$t=0$



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